

## VIBRATIONS

Sushil Kumar  
Singh

Outline

Simple Harmonic  
Motion

Linearity of EOM  
⇒ Superposition

Nature of Motion

Energy of the  
System

Examples

# VIBRATIONS

## Unit 1 *Week 1*

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- The behaviour of displacement with time has a sinusoidal or cosinusoidal dependence.

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- The behaviour of displacement with time has a sinusoidal or cosinusoidal dependence.
- The maximum displacement on either side of the equilibrium is the same.

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- The behaviour of displacement with time has a sinusoidal or cosinusoidal dependence.
- The maximum displacement on either side of the equilibrium is the same.
- The energy is conserved and the oscillation continues forever.

# EXAMPLES

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- a simple pendulum, a mass  $m$  swinging at the end of a light rigid rod of length  $l$

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- a simple pendulum, a mass  $m$  swinging at the end of a light rigid rod of length  $l$
- a frictionless U-tube of constant cross-sectional area containing a length  $l$  of liquid, density  $\rho$ , oscillating about its equilibrium position of equal levels in each limb

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- a frictionless U-tube of constant cross-sectional area containing a length  $l$  of liquid, density  $\rho$ , oscillating about its equilibrium position of equal levels in each limb
- an electrical circuit, an inductance  $L$  connected across a capacitance  $C$  carrying a charge  $q$



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- A small displacement  $\vec{x}$  from its equilibrium position sets up a restoring force which is proportional to  $x$  acting in a direction towards the equilibrium position.

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- A small displacement  $\vec{x}$  from its equilibrium position sets up a restoring force which is proportional to  $x$  acting in a direction towards the equilibrium position.
- Thus, this restoring force  $\vec{F}$  may be written

$$\vec{F} = -k\vec{x} \quad (1)$$

where  $k$ , the constant of proportionality, is called the *stiffness* and the negative sign shows that the force is acting *against* the direction of increasing displacement and back towards the equilibrium position. This is **Hooke's Law of Elasticity**.

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- If the stiffness  $k$  is constant, then the restoring force  $F$  when plotted versus  $x$ , will produce a straight line and the system is said to be *linear*.

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- If the stiffness  $k$  is constant, then the restoring force  $F$  when plotted versus  $x$ , will produce a straight line and the system is said to be *linear*.
- The equation of motion of such a disturbed system is given by the *dynamic balance* between the forces acting on the system, which by **Newton's Law** is

$$m\ddot{\vec{x}} = -k\vec{x} \quad (2)$$

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The equation being linear we find that if  $X_1$  and  $X_2$  are two different solutions then

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The equation being linear we find that if  $X_1$  and  $X_2$  are two different solutions then

$$m\ddot{\vec{X}}_1 = -k\vec{X}_1$$

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$$m\ddot{\vec{X}}_1 = -k\vec{X}_1$$

$$m\ddot{\vec{X}}_2 = -k\vec{X}_2$$

$$m\ddot{\vec{X}}_1 + m\ddot{\vec{X}}_2 = (-k\vec{X}_1) + (-k\vec{X}_2)$$



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$$m\ddot{\vec{X}}_1 + m\ddot{\vec{X}}_2 = (-k\vec{X}_1) + (-k\vec{X}_2)$$

$$m \frac{d^2(\vec{X}_1 + \vec{X}_2)}{dt^2} = -k(\vec{X}_1 + \vec{X}_2)$$

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Examples

- We find that

$$x = A \cos \omega t$$

and

$$x = B \sin \omega t$$

are independent solutions with

$$\omega = \sqrt{\frac{k}{m}}$$

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- We find that

$$x = A \cos \omega t$$

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- The complete or general solution is given by the superposition of both values for  $x$  so we have

$$x = A \cos \omega t + B \sin \omega t$$

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- If we rewrite the constants as  $A = a \sin \phi$  and  $B = a \cos \phi$  where  $\phi$  is a constant angle, then  $a = \sqrt{A^2 + B^2}$  and

$$x = a \sin \phi \cos \omega t + a \cos \phi \sin \omega t = a \sin(\omega t + \phi) \quad (3)$$

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Examples

- The system will oscillate with *Time Period*  $T = 2\pi/\omega$  between the values of  $x = \pm a$  so that  $a$  is the *Amplitude* of displacement.

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- The system will oscillate with *Time Period*  $T = 2\pi/\omega$  between the values of  $x = \pm a$  so that  $a$  is the *Amplitude* of displacement.
- The magnitude of  $a$  is determined by the total energy of the oscillator.

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- The angle  $\phi$  is called the *Phase constant*. The inclusion of  $\phi$  in the solution allows the motion to be defined from any starting point.

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- The magnitude of  $a$  is determined by the total energy of the oscillator.
- The angle  $\phi$  is called the *Phase constant*. The inclusion of  $\phi$  in the solution allows the motion to be defined from any starting point.
- The velocity of the oscillating system is

$$\dot{x} = \omega a \cos(\omega t + \phi) = \omega a \sin(\omega t + \phi + \pi/2)$$

while the acceleration is

$$\ddot{x} = -\omega^2 a \sin(\omega t + \phi) = \omega^2 a \sin(\omega t + \phi + \pi)$$



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- The fact that the velocity is zero at maximum displacement in simple harmonic motion and is a maximum at zero displacement illustrates the important concept of an exchange between kinetic and potential energy.

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- The fact that the velocity is zero at maximum displacement in simple harmonic motion and is a maximum at zero displacement illustrates the important concept of an exchange between kinetic and potential energy.
- The potential energy at a position  $x$  is found by summing all the small elements of work done by the system against the restoring force over the range zero to  $x$  where  $x = 0$  gives zero potential energy

$$U = \int -(-k\vec{x}) \cdot d\vec{x} = \int_0^x kx dx = \frac{1}{2}kx^2$$

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- The kinetic energy at that position  $x$  is

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$$U = \int -(-k\vec{x}) \cdot d\vec{x} = \int_0^x kx dx = \frac{1}{2} kx^2$$

- The kinetic energy at that position  $x$  is

$$K = \frac{1}{2} m\dot{x}^2$$

- The total energy at any position  $x$

$$E = U + K = \frac{1}{2} kx^2 + \frac{1}{2} m\dot{x}^2 = \frac{1}{2} ka^2 \quad (4)$$

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- The maximum potential energy is  $U_{max} = \frac{1}{2}ka^2$

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- The maximum potential energy is  $U_{max} = \frac{1}{2}ka^2$
- The maximum kinetic energy is  $K_{max} = \frac{1}{2}m\dot{x}^2 = \frac{1}{2}m\omega^2a^2 = \frac{1}{2}ka^2$

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- The maximum kinetic energy is  $K_{max} = \frac{1}{2}m\dot{x}^2 = \frac{1}{2}m\omega^2a^2 = \frac{1}{2}ka^2$
- The average kinetic energy  $\langle K \rangle$  and the average potential energy  $\langle U \rangle$  over a time period  $T$

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- The average kinetic energy  $\langle K \rangle$  and the average potential energy  $\langle U \rangle$  over a time period  $T$

$$\begin{aligned}
 \langle K \rangle &= \frac{\oint_T \frac{1}{2}m\dot{x}^2 dt}{\oint_T dt} \\
 &= \frac{\int_T \frac{1}{2}m\omega^2a^2 \cos^2(\omega t + \phi) dt}{T} \\
 &= \frac{\int_T \frac{1}{2}m\omega^2a^2 \cos^2 \theta d\theta}{\omega T} \\
 &= \frac{1}{2}m\omega^2a^2 \frac{\int_T \cos^2 \theta d\theta}{2\pi} \\
 &= \frac{1}{2}ka^2 \frac{(4 \int_0^{\pi/2} \cos^2 \theta d\theta)}{2\pi} \\
 &= \frac{1}{4}ka^2 = \frac{1}{2}E
 \end{aligned}$$



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- The average kinetic energy  $\langle K \rangle$  and the average potential energy  $\langle U \rangle$  over a time period  $T$

$$\begin{aligned}
 \langle K \rangle &= \frac{\oint_T \frac{1}{2}m\dot{x}^2 dt}{\oint_T dt} & \langle U \rangle &= \frac{\oint_T \frac{1}{2}kx^2 dt}{\oint_T dt} \\
 &= \frac{\int_T \frac{1}{2}m\omega^2a^2 \cos^2(\omega t + \phi) dt}{T} & &= \frac{\int_T \frac{1}{2}ka^2 \sin^2(\omega t + \phi) dt}{T} \\
 &= \frac{\int_T \frac{1}{2}m\omega^2a^2 \cos^2 \theta d\theta}{\omega T} & &= \frac{\int_T \frac{1}{2}ka^2 \sin^2 \theta d\theta}{\omega T} \\
 &= \frac{1}{2}m\omega^2a^2 \frac{\int_T \cos^2 \theta d\theta}{2\pi} & &= \frac{1}{2}ka^2 \frac{\int_T \sin^2 \theta d\theta}{2\pi} \\
 &= \frac{1}{2}ka^2 \frac{(4 \int_0^{\pi/2} \cos^2 \theta d\theta)}{2\pi} & &= \frac{1}{2}ka^2 \frac{(4 \int_0^{\pi/2} \sin^2 \theta d\theta)}{2\pi} \\
 &= \frac{1}{4}ka^2 = \frac{1}{2}E & &= \frac{1}{4}ka^2 = \frac{1}{2}E
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- a simple pendulum  $(ml^2)\ddot{\theta} = \tau = \vec{l} \times \vec{F} = lmg \sin \theta(-\hat{\theta})$  with  $\omega^2 = g/l$

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- a frictionless U-tube  $(lA\rho)\ddot{x} = -[(2x)A\rho]g$  with  $\omega^2 = 2g/l$

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- a frictionless U-tube  $(lA\rho)\ddot{x} = -[(2x)A\rho]g$  with  $\omega^2 = 2g/l$
- an LC circuit  $V_L(= -Ldi/dt = -Ld^2q/dt^2) = V_C(= q/C)$  with  $\omega^2 = 1/LC$